

# Dynamical systems and Control

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# Lecture 3: State-space representation

- Recap
- Transfer function of state-space model
- Controllability and observability
- Linearization

# Characteristics of first and second order systems

A quick recap of lecture 2

# First-order system

$$T\dot{y}(t) + y(t) = Ku(t)$$

$$G(s) = \frac{Y(s)}{U(s)} = \frac{K}{Ts + 1}$$

- $T$ : Time constant
- $K$ : Static gain
- The pole is  $\lambda = -\frac{1}{T}$

# First-order system

$$T\dot{y}(t) + y(t) = Ku(t)$$

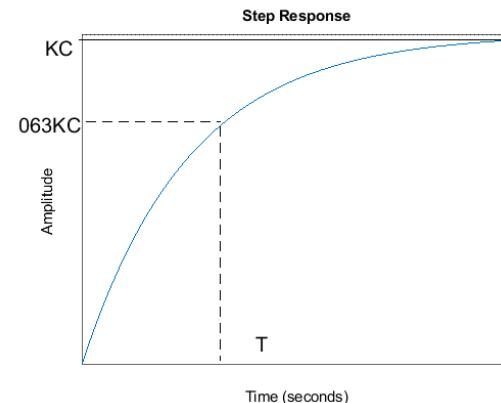
Take Laplace:

$$TsY(s) + Y(s) = KU(s)$$



$$G(s) = \frac{Y(s)}{U(s)} = \frac{K}{Ts + 1}$$

- $T$ : Time constant- The time the output reaches 63% of its final value
- $K$ : Static gain: How much the input is magnified in the steady state
- The pole is  $\lambda = -\frac{1}{T}$



# Second-order system

$$G(s) = \frac{a_2}{s^2 + a_1 s + a_2}$$

Complex poles

$$G(s) = K \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2} \quad \xrightarrow{\hspace{1cm}} \quad \lambda_{1,2} = \omega_0(-\zeta \pm i\sqrt{1 - \zeta^2})$$

# Example

$$G(s) = \frac{4}{s^2 + 5s + 9}$$

# Example

$$G(s) = \frac{4}{s^2 + 5s + 9}$$

$$G(s) = \frac{4}{9} \frac{3^2}{s^2 + 2 \times \frac{5}{6} \times 3s + 3^2}$$

Compare with

$$G(s) = K \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

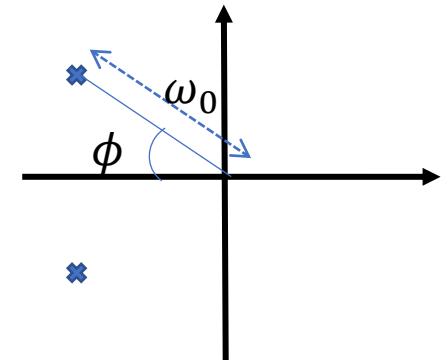
Then, we have

$$K = \frac{4}{9} \quad \zeta = \frac{5}{6} \quad \omega_0 = 3$$

# Second-order system with complex poles

$$G(s) = \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

Complex poles:  $\lambda_{1,2} = \omega_0(-\zeta \pm i\sqrt{1 - \zeta^2})$



# Second-order system with complex poles

$$G(s) = \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

Complex poles:  $\lambda_{1,2} = \omega_0(-\zeta \pm i\sqrt{1 - \zeta^2})$

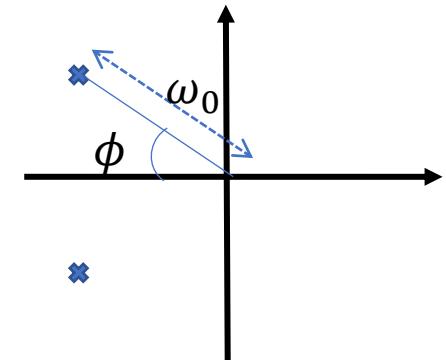
Write the pole in polar system:

$\omega_0$  = Distance to the origin

$$\sqrt{\omega_0^2\zeta^2 + \omega_0^2(1 - \zeta^2)} = \omega_0$$

$\zeta$  = relative damping

$$\cos \phi = \frac{\omega_0 \zeta}{\omega_0} = \zeta$$



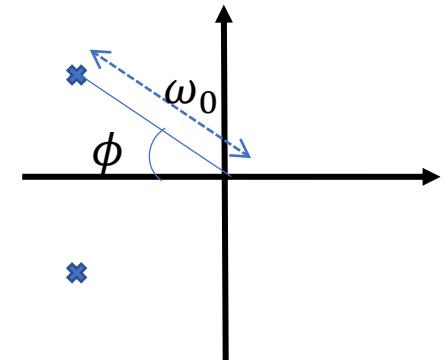
# Second-order system with complex poles

$$G(s) = \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

Complex poles:  $\lambda_{1,2} = \omega_0(-\zeta \pm i\sqrt{1 - \zeta^2})$

Step response:

$$y(t) = 1 - \frac{e^{-\zeta\omega_0 t}}{\sqrt{1 - \zeta^2}} \sin(\omega_0 \sqrt{1 - \zeta^2} t + \phi)$$



# Second-order system with complex poles

$$G(s) = \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

Complex poles:  $\lambda_{1,2} = \omega_0(-\zeta \pm i\sqrt{1 - \zeta^2})$

Step response:

$$Y(s) = \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2} \frac{1}{s} = \frac{1}{s} - \frac{s+2\zeta\omega_0}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

Take inverse Laplace

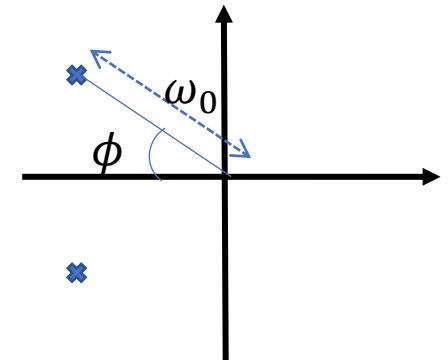
$$y(t) = 1 - \mathcal{L}^{-1} \left\{ \frac{s + 2\zeta\omega_0}{s^2 + 2\zeta\omega_0 s + \omega_0^2} \right\}$$

Use A.27 on p. 233, let  $a = \zeta\omega_0$ ,  $b = \omega_0\sqrt{1 - \zeta^2}$ ,  $z = 2\zeta\omega_0$

$$y(t) = 1 - \frac{e^{-\zeta\omega_0 t}}{\sqrt{1 - \zeta^2}} (\sqrt{1 - \zeta^2} \cos \omega_0 \sqrt{1 - \zeta^2} t + \zeta \sin \omega_0 \sqrt{1 - \zeta^2} t)$$

Introduce  $\zeta = \cos \phi$ ,  $\omega_d = \omega_0 \sqrt{1 - \zeta^2}$

$$y(t) = 1 - \frac{e^{-\zeta\omega_0 t}}{\sqrt{1 - \zeta^2}} (\sin \phi \cos \omega_d t + \cos \phi \sin \omega_d t) = 1 - \frac{e^{-\zeta\omega_0 t}}{\sqrt{1 - \zeta^2}} \sin(\omega_d t + \phi)$$



# Second-order system with complex poles

$$y(t) = 1 - \frac{e^{-\zeta\omega_0 t}}{\sqrt{1-\zeta^2}} \sin(\omega_0 \sqrt{1-\zeta^2} t + \phi)$$

$\zeta$ : relative damping:

$\omega_0$ : kind of time scaling

# Second-order system with complex poles

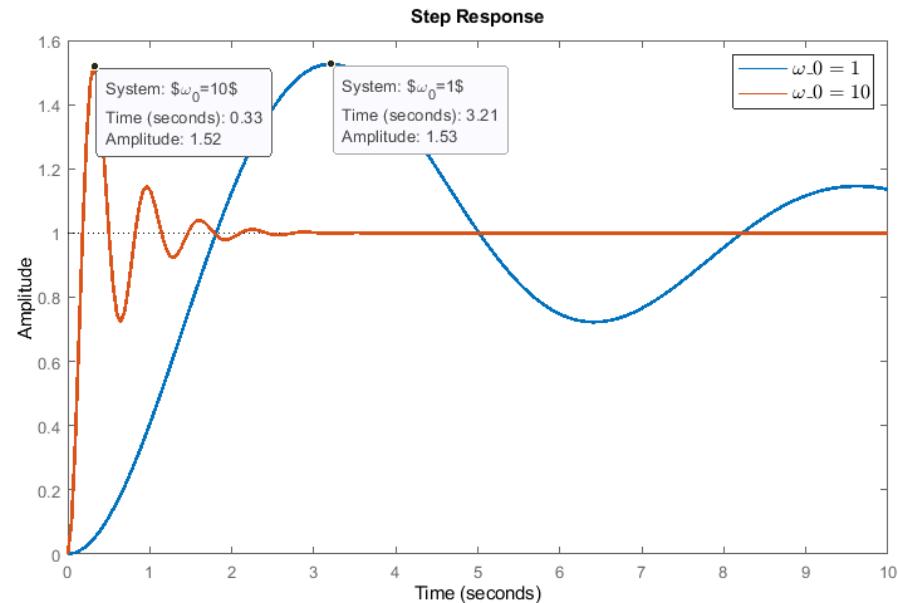
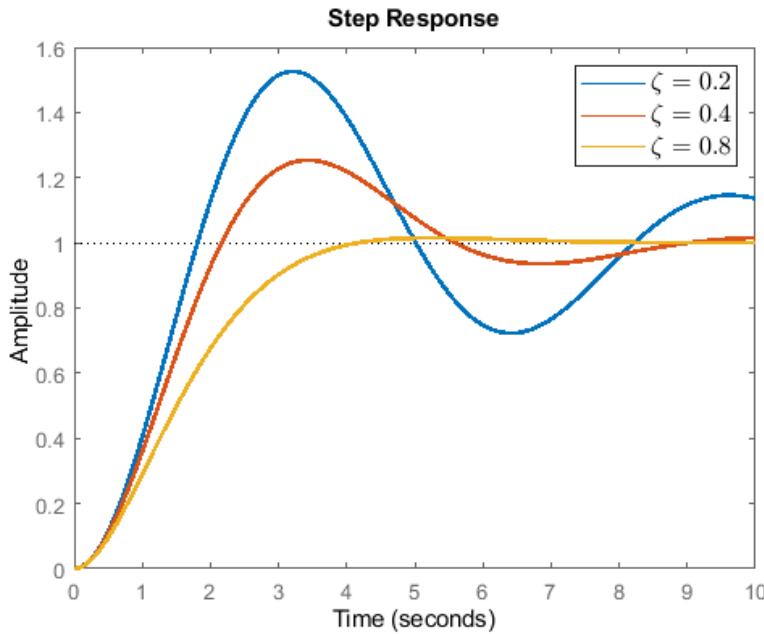
$$y(t) = 1 - \frac{e^{-\zeta\omega_0 t}}{\sqrt{1-\zeta^2}} \sin(\omega_0 \sqrt{1-\zeta^2} t + \phi)$$

$\zeta$ : relative damping:

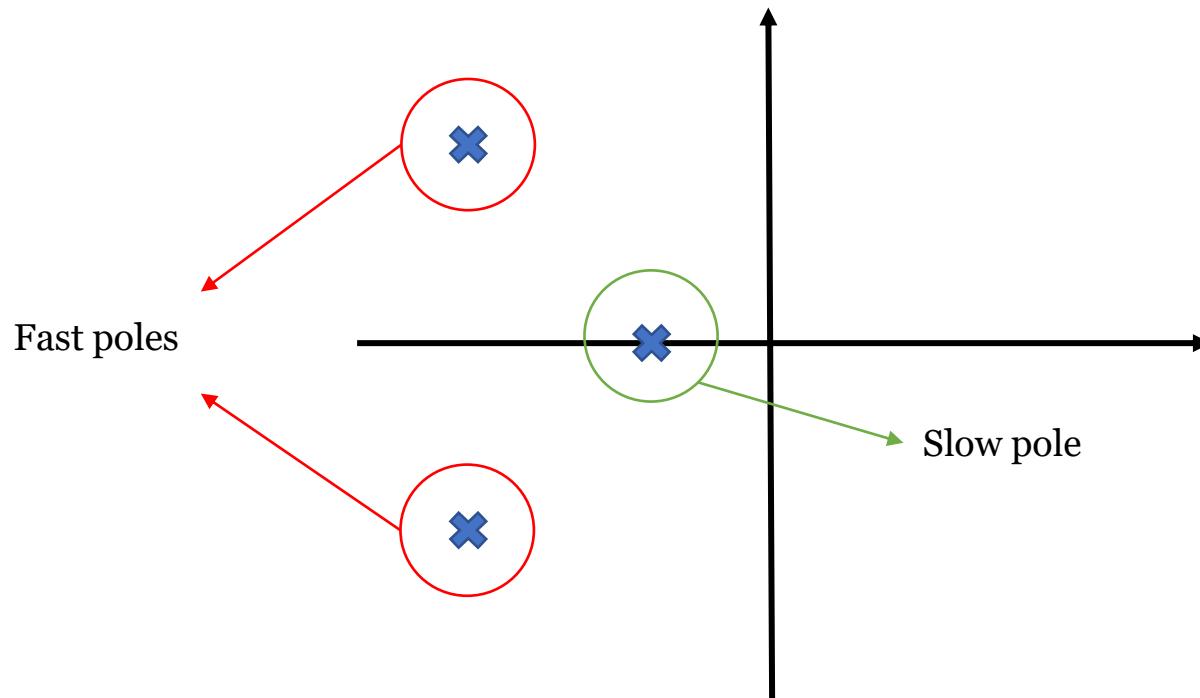
- **Small:** weak damping (more oscillations)
- **Big:** strong damping (less oscillations)

$\omega_0$ : kind of time scaling

- E.g. 10 times bigger, 10 times faster



# Fast and slow dynamics



# Transfer function of state-space model

# State-space model



$$\dot{x} = Ax + Bu \quad (1)$$

$$y = Cx + Du \quad (2)$$

$$G(s) = C(sI - A)^{-1}B + D$$

# State-space model



$$\dot{x} = Ax + Bu \quad (1)$$

$$y = Cx + Du \quad (2)$$

First, note that

$$\mathcal{L}\{\dot{x}\} = \begin{bmatrix} \mathcal{L}\{\dot{x}_1\} \\ \vdots \\ \mathcal{L}\{\dot{x}_n\} \end{bmatrix} = \begin{bmatrix} sX_1(s) \\ \vdots \\ sX_n(s) \end{bmatrix} = \begin{bmatrix} s & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & s \end{bmatrix} X(s) = sI_n X(s)$$

We take the Laplace transformation of (1)

$$sI_n X(s) = AX(s) + BU(s) \rightarrow X(s) = (sI - A)^{-1}BU(s)$$

Now, we replace this result in Laplace of (2)

$$Y(s) = CX(s) + DU(s) = (C(sI - A)^{-1}B + D)U(s)$$

$$G(s) = C(sI - A)^{-1}B + D$$

# Example- What is the transfer function for Boeing 747

$$\begin{bmatrix} \dot{\alpha} \\ \dot{q} \end{bmatrix} = \begin{bmatrix} -0.39 & 1 \\ -1.55 & -0.53 \end{bmatrix} \begin{bmatrix} \alpha \\ q \end{bmatrix} + \begin{bmatrix} -18.5 \\ -1.2 \end{bmatrix} \delta_e$$

$\alpha$ : Angle of attack

$q$ : Pitch rotation rate

$$y = [0 \quad 1] \begin{bmatrix} \alpha \\ q \end{bmatrix}$$

**Solution:**

# Example- What is the transfer function for Boeing 747

$$\begin{bmatrix} \dot{\alpha} \\ \dot{q} \end{bmatrix} = \begin{bmatrix} -0.39 & 1 \\ -1.55 & -0.53 \end{bmatrix} \begin{bmatrix} \alpha \\ q \end{bmatrix} + \begin{bmatrix} -18.5 \\ -1.2 \end{bmatrix} \delta_e$$

$$y = [0 \quad 1] \begin{bmatrix} \alpha \\ q \end{bmatrix}$$

Solution:

$$(sI - A)^{-1} = \begin{bmatrix} s + 0.39 & -1 \\ 1.55 & s + 0.53 \end{bmatrix}^{-1} = \frac{1}{(s+0.39)(s+0.53)+1.55} \begin{bmatrix} s + 0.53 & 1 \\ -1.55 & s + 0.39 \end{bmatrix}$$

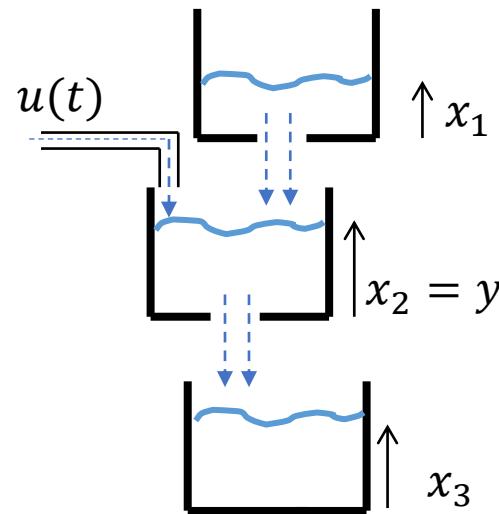
$$G(s) = \frac{1}{(s + 0.39)(s + 0.53) + 1.55} [0 \quad 1] \begin{bmatrix} s + 0.53 & 1 \\ -1.55 & s + 0.39 \end{bmatrix} \begin{bmatrix} -18.5 \\ -1.2 \end{bmatrix}$$

$$G(s) = \frac{-1.2s + 28.2}{s^2 + 0.92s + 1.7567}$$

Use `ss2tf(A, B, C, D)` in MATLAB!

# Controllability and Observability

# Example- A triple-tank



# Controllability

**Definition:** The system is controllable if there exists an input signal that brings the state of the system to any desired  $x^*$  from the origin.

**Method:** The system is controllable if

$$\text{rank}([B \ AB \ \dots \ A^{n-1}B] = n)$$

# Observability

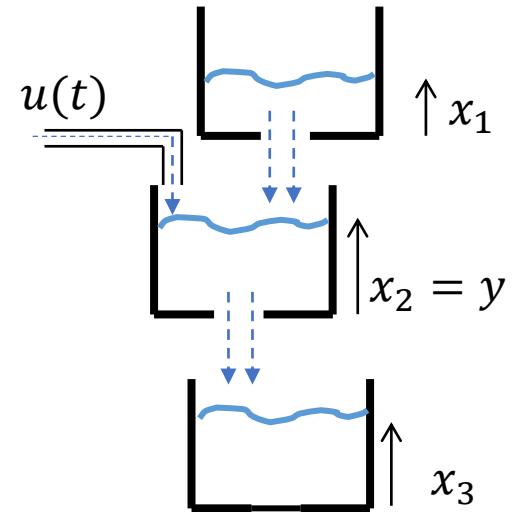
**Definition:** Let  $u = 0$ . If  $x^*(0) \neq 0$  leads to  $y(t) = 0$ , we say that  $x^*$  is unobservable. If there is no unobservable state, then the system is observable

**Method:** The system is observable if

$$\text{rank} \begin{bmatrix} C \\ CA \\ \vdots \\ CA^{n-1} \end{bmatrix} = n)$$

# Example- Is the triple-tank controllable and observable?

**Solution:**



# Example- Is the triple-tank controllable and observable?

**Solution:** write differential equations

$$\dot{x}_1 = -x_1$$

$$\dot{x}_2 = x_1 - x_2 + u \quad y = x_2$$

$$\dot{x}_3 = x_2$$

Now, write them in state space format:

$$\dot{x} = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -1 & 0 \\ 0 & 1 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} u \quad y = [0 \ 1 \ 0] x$$

Controllability:

$$[B, AB, A^2B] = \begin{bmatrix} 0 & 0 & 0 \\ 1 & -1 & 1 \\ 0 & 1 & -1 \end{bmatrix} \rightarrow \text{Det}([B, AB, A^2B]) = 0$$

Not controllable

Observability:

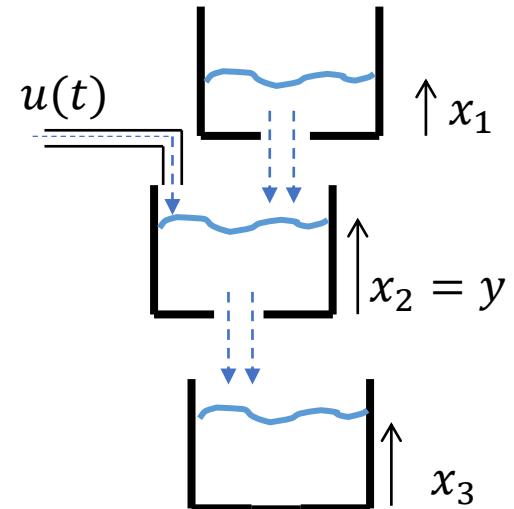
$$\begin{bmatrix} C \\ CA \\ CA^2 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 1 & -1 & 0 \\ -2 & 1 & 0 \end{bmatrix} \rightarrow \text{Det}\left(\begin{bmatrix} C \\ CA \\ CA^2 \end{bmatrix}\right) = 0$$

Not observable

Transfer function:

$$G = \frac{1}{s+1}$$

Two poles are missing!



# Linearization

# Nonlinear model

$$\dot{x}(t) = f(x(t), u(t))$$

$$y(t) = h(x(t), u(t))$$

Stationary point:

$$f(x_0, u_0) = 0$$

$$y_0 = h(x_0, u_0)$$

# Linearized model

Let

$$\Delta x = x - x_0$$

$$\Delta u = u - u_0$$

$$\Delta y = y - y_0$$

Linearized model:

$$\dot{\Delta x} = A\Delta x + B\Delta u$$

$$\Delta y = C\Delta x + D\Delta u$$

$$A = \frac{\partial f}{\partial x} \Big|_{x=x_0, u=u_0}$$

$$C = \frac{\partial h}{\partial x} \Big|_{x=x_0, u=u_0}$$

$$B = \frac{\partial f}{\partial u} \Big|_{x=x_0, u=u_0}$$

$$D = \frac{\partial h}{\partial u} \Big|_{x=x_0, u=u_0}$$

# Linearized model

Let

$$\Delta x = x - x_0$$

$$\Delta u = u - u_0$$

$$\Delta y = y - y_0$$

Derive  $\dot{x}$  and use Taylor expansion

$$\begin{aligned}\dot{x} &= \dot{x} - \dot{x}_0 = f(x, u) = f(x_0 + \Delta x, u_0 + \Delta u) \approx f(x_0, u_0) + \frac{\partial f}{\partial x} \Big|_{x=x_0, u=u_0} \Delta x + \frac{\partial f}{\partial u} \Big|_{x=x_0, u=u_0} \Delta u \\ &\approx \frac{\partial f}{\partial x} \Big|_{x=x_0, u=u_0} \Delta x + \frac{\partial f}{\partial u} \Big|_{x=x_0, u=u_0} \Delta u\end{aligned}$$

Write  $\Delta y$

$$\begin{aligned}\Delta y &= h(x, u) - y_0 = h(x_0 + \Delta x, u_0 + \Delta u) - y_0 \approx h(x_0, u_0) + \frac{\partial h}{\partial x} \Big|_{x=x_0, u=u_0} \Delta x + \frac{\partial h}{\partial u} \Big|_{x=x_0, u=u_0} \Delta u - y_0 \\ &\approx \frac{\partial h}{\partial x} \Big|_{x=x_0, u=u_0} \Delta x + \frac{\partial h}{\partial u} \Big|_{x=x_0, u=u_0} \Delta u\end{aligned}$$

Linearized model:

$$\dot{x} = A\Delta x + B\Delta u$$

$$A = \frac{\partial f}{\partial x} \Big|_{x=x_0, u=u_0}$$

$$B = \frac{\partial f}{\partial u} \Big|_{x=x_0, u=u_0}$$

$$\Delta y = C\Delta x + D\Delta u$$

$$C = \frac{\partial h}{\partial x} \Big|_{x=x_0, u=u_0}$$

$$D = \frac{\partial h}{\partial u} \Big|_{x=x_0, u=u_0}$$

# Linearized model

$n$  states,  $m$  inputs,  $p$  outputs

$$A = \frac{\partial f}{\partial x} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial f_1}{\partial x_1} & \dots & \frac{\partial f_1}{\partial x_n} \\ \vdots & \ddots & \vdots \\ \frac{\partial f_n}{\partial x_1} & \dots & \frac{\partial f_n}{\partial x_n} \end{bmatrix}$$

$$B = \frac{\partial f}{\partial u} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial f_1}{\partial u_1} & \dots & \frac{\partial f_1}{\partial u_m} \\ \vdots & \ddots & \vdots \\ \frac{\partial f_n}{\partial u_1} & \dots & \frac{\partial f_n}{\partial u_m} \end{bmatrix}$$

$$C = \frac{\partial h}{\partial x} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial h_1}{\partial x_1} & \dots & \frac{\partial h_1}{\partial x_n} \\ \vdots & \ddots & \vdots \\ \frac{\partial h_p}{\partial x_1} & \dots & \frac{\partial h_p}{\partial x_n} \end{bmatrix}$$

$$D = \frac{\partial h}{\partial u} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial h_1}{\partial u_1} & \dots & \frac{\partial h_1}{\partial u_m} \\ \vdots & \ddots & \vdots \\ \frac{\partial h_p}{\partial u_1} & \dots & \frac{\partial h_p}{\partial u_m} \end{bmatrix}$$

# Linearized model

$n$  states,  $m$  inputs,  $p$  outputs

$$f(x, u) = \begin{bmatrix} f_1(x, u) \\ \vdots \\ f_n(x, u) \end{bmatrix} \quad h(x, u) = \begin{bmatrix} h_1(x, u) \\ \vdots \\ h_p(x, u) \end{bmatrix}$$

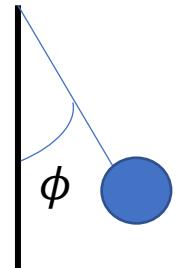
$$A = \frac{\partial f}{\partial x} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial f_1}{\partial x_1} & \dots & \frac{\partial f_1}{\partial x_n} \\ \vdots & \ddots & \vdots \\ \frac{\partial f_n}{\partial x_1} & \dots & \frac{\partial f_n}{\partial x_n} \end{bmatrix} \quad B = \frac{\partial f}{\partial u} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial f_1}{\partial u_1} & \dots & \frac{\partial f_1}{\partial u_m} \\ \vdots & \ddots & \vdots \\ \frac{\partial f_n}{\partial u_1} & \dots & \frac{\partial f_n}{\partial u_m} \end{bmatrix}$$

$$C = \frac{\partial h}{\partial x} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial h_1}{\partial x_1} & \dots & \frac{\partial h_1}{\partial x_n} \\ \vdots & \ddots & \vdots \\ \frac{\partial h_p}{\partial x_1} & \dots & \frac{\partial h_p}{\partial x_n} \end{bmatrix} \quad D = \frac{\partial h}{\partial u} \Big|_{x=x_0, u=u_0} = \begin{bmatrix} \frac{\partial h_1}{\partial u_1} & \dots & \frac{\partial h_1}{\partial u_m} \\ \vdots & \ddots & \vdots \\ \frac{\partial h_p}{\partial u_1} & \dots & \frac{\partial h_p}{\partial u_m} \end{bmatrix}$$

# Example- Linearize the model

$$J\ddot{\phi} = -mg \sin \phi - c\dot{\phi}$$

**Solution:**



# Example- Linearize the model

$$J\ddot{\phi} = -mg \sin \phi - c\dot{\phi}$$

**Solution:** Let

$$x_1 = \phi \quad x_2 = \dot{\phi}$$

Write the space state equations:

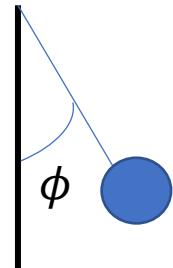
$$\dot{x}_1 = x_2 \quad \dot{x}_2 = -\frac{mg}{J} \sin x_1 - \frac{c}{J} x_2$$

Stationary point by  $\dot{x}_1 = 0, \dot{x}_2 = 0$ :

case1)  $x_{20} = 0, x_{10} = 0$  or

case 2)  $x_{20} = 0, x_{10} = \pi$

$$\frac{\partial f}{\partial x} = \begin{bmatrix} 0 & 1 \\ -\frac{mg}{J} \cos x_1 & -\frac{c}{J} \end{bmatrix}$$



case1)  $x_{20} = 0, x_{10} = 0$

$$A_1 = \frac{\partial f}{\partial x} \Big|_{x=x_0} = \begin{bmatrix} 0 & 1 \\ -\frac{mg}{J} & -\frac{c}{J} \end{bmatrix}$$

stable

case2)  $x_{20} = 0, x_{10} = \pi$

$$A_1 = \frac{\partial f}{\partial x} \Big|_{x=x_0} = \begin{bmatrix} 0 & 1 \\ \frac{mg}{J} & -\frac{c}{J} \end{bmatrix}$$

unstable

# What do we cover next?

- Sensor
- Introduction to signal processing

# Ask us!

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